

Vibration of Visco-Elastic Parallelogram Plate with Thickness Variation Linearly in one Direction and Parabolic in another Direction

Arun Kumar Gupta*, Anuj Kumar

Department of Mathematics, M. S. College, Saharanpur, U.P., India

*Corresponding author: gupta_arunnitin@yahoo.co.in

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Abstract The main objective of the present investigation is to study the vibration of visco-elastic parallelogram plate whose thickness varies bi-directionally. It is assumed that the plate is clamped on all the four edges and that the thickness varies linearly in one direction and parabolically in another direction. Using the separation of variables method and Rayleigh-Ritz technique with a two-term deflection function, the governing differential equation has been solved for vibration of visco-elastic parallelogram plate. For visco-elastic, the basic elastic and viscous elements are combined. We have taken Kelvin model for visco-elasticity that is the combination of the elastic and viscous elements in parallel. Here the elastic element means the spring and the viscous element means the dashpot. The assumption of small deflection is made. Visco-elastic of the plate is taken of the “Kelvin Type”. Time period and deflection function at different point for the first two modes of vibration are calculated for various values of taper constant, aspect ratio and skew angle and results are presented in tabular form. Alloy “Duralumin” is considered for all the material constants used in numerical calculations.

Keywords: vibration, parallelogram plate, visco-elastic mechanics, linear thickness variation, parabolic thickness variation, both directions

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1. Introduction

Leissa [1,2,3] discussed the recent studies in plate vibration. Sobotka [4] used analysis of thick visco-elastic plates. The theory of linear visco-elasticity has been discussed by Bland [5]. Ilanko [6] solved comments on the historical bases of the Rayleigh and Ritz methods. Eslami et al. [7] discussed coupled thermo elasticity of shell of revolution effect of normal stress and coupling. Leissa and Narita [8] discussed vibration studies for simply supported symmetrically laminated rectangular plates.

Lekhnitski [9] analyzed anisotropic plates. Zhang and Zu [10] solved non-linear vibrations of visco-elastic moving belts part-I: force vibration analysis. Zhang and Zu [11] used non linear vibration of parametrically excited visco-elastic moving belts part-II stability analysis. Mivhel [12] solved a periodic problem in visco-elasticity with variable coefficients. The survey of aero thermo-elasticity is discussed by Garrick [13]. Gnossi and Laura [14] have discussed transverse vibrations of rectangular orthotropic plates with one or two free edges while the remaining is elastically restrained against rotation. Park and Mongeau [15] used the vibration and radiation of visco elastically supported midline plates. Gupta et al. [16]

study the vibration of visco-elastic parallelogram plate of linearly varying thickness.

A simple model presented here is to study the effect of linearly thickness variation in one direction and parabolic in another direction on vibration of visco-elastic parallelogram plate having clamped boundary conditions on all the four edges. The hypothesis of small deflection and linear visco-elastic properties are made. Using the separation of variables method, the governing differential equation has been solved for vibration of visco-elastic parallelogram plate. An approximate but quite convenient frequency equation is derived by using Rayleigh-Ritz technique with a two-term deflection function. It is assumed that the visco-elastic of the plate is of the “Kelvin Type”. Time period and deflection function at different point for the first two modes of vibration are calculated for various values of taper constant, aspect ratio, and skew angle and results are presented in tabular and graphically form.

2. Equation of Transverse Motion

The maximum kinetic energy, T_{max} , and the maximum strain energy, V in the plate when it is executing transverse vibration mode shape $W(\xi, \eta)$ in skew coordinates are [16]:

$$T_{\max} = \frac{1}{2} \rho p^2 \cos \theta \iint h W^2 d\xi d\eta \quad (1)$$

and

$$V = \frac{1}{2} \cos \theta \iint D \left[(\nabla^2 W)^2 - 2(1-\nu) \sec^2 \theta (W_{,\xi\xi} W_{,\eta\eta} - W_{,\xi\eta}^2) \right] d\xi d\eta$$

or

$$V = \frac{1}{2 \cos^3 \theta} \iint D \left[W_{,\xi\xi}^2 - 4 \sin \theta W_{,\xi\xi} W_{,\xi\eta} + 2(\sin^2 \theta + \nu \cos^2 \theta) W_{,\xi\xi} W_{,\eta\eta} + 2(1 + \sin^2 \theta - \nu \cos^2 \theta) W_{,\xi\eta}^2 - 4 \sin \theta W_{,\xi\eta} W_{,\eta\eta} + W_{,\eta\eta}^2 \right] d\xi d\eta \quad (2)$$

and

$$T_{,tt} + p^2 \ddot{T} = 0 \quad (3)$$

A comma followed by a suffix denotes partial differential with respect to that variable. Here p^2 is a constant.

Here solution $w(\xi, \eta, t)$ can be taken in the form of products of two functions as for free transverse vibration of the parallelogram plate so that

$$w(\xi, \eta, t) = W(\xi, \eta) T(t) \quad (4)$$

where $T(t)$ is the time function and W is the maximum displacement with respect to time t .

Assuming thickness variation of parallelogram plate linearly in ξ -direction and parabolically in η -direction, as

$$h = h_0 \{1 + \beta_1 (\xi/a)\} \{1 + \beta_2 (\eta/b)^2\} \quad (5)$$

where β_1 and β_2 is the taper constants in ξ and η - directions respectively and $h_0 = h|_{\xi=\eta=0}$.

The flexural rigidity of the plate can now be written as

$$D = Eh_0^3 \{1 + \beta_1 (\xi/a)\}^3 \{1 + \beta_2 (\eta/b)^2\}^3 / 12(1-\nu^2) \quad (6)$$

3. Solution and Frequency Equation

In using the Rayleigh-Ritz technique, one requires maximum strain energy be equal to the maximum kinetic energy. So it is necessary for the problem consider here that

$$\delta(V - T_{\max}) = 0 \quad (7)$$

for arbitrary variations of W satisfying relevant geometrical boundary conditions.

For a parallelogram plate, clamped (c) along all the four edges, the boundary conditions are

$$\begin{aligned} W = W_{,\xi} = 0 \quad \text{at } \xi = 0, a \\ \text{and} \\ W = W_{,\eta} = 0 \quad \text{at } \eta = 0, b \end{aligned} \quad (8)$$

and the corresponding two-term deflection function is taken as

$$W = \left[(\xi/a)(\eta/b)(1-\xi/a)(1-\eta/b) \right]^2 \times [A_1 + A_2 (\xi/a)(\eta/b)(1-\xi/a)(1-\eta/b)] \quad (9)$$

Using equation (5) and (6) in equations (1) and (2) one obtains

$$T_{\max} = \frac{1}{2} h_0 \rho p^2 \cos \theta \int_{\eta=0}^b \int_{\xi=0}^a (1 + \beta_1 (\xi/a))(1 + \beta_2 (\eta/b)^2) W^2 d\xi d\eta \quad (10)$$

and

$$V = \frac{Eh_0^3}{24(1-\nu^2) \cos^3 \theta} \int_{\eta=0}^b \int_{\xi=0}^a (1 + \beta_1 (\xi/a))^3 (1 + \beta_2 (\eta/b)^2)^3 \left[W_{,\xi\xi}^2 - 4 \left(\frac{a}{b} \right) \sin \theta W_{,\xi\xi} W_{,\xi\eta} + 2 \left(\frac{a}{b} \right)^2 (\sin^2 \theta + \nu \cos^2 \theta) W_{,\xi\xi} W_{,\eta\eta} + 2 \left(\frac{a}{b} \right)^2 (1 + \sin^2 \theta - \nu \cos^2 \theta) W_{,\xi\eta}^2 - 4 \left(\frac{a}{b} \right)^3 \sin \theta W_{,\xi\eta} W_{,\eta\eta} + \left(\frac{a}{b} \right)^4 W_{,\eta\eta}^2 \right] d\xi d\eta \quad (11)$$

Using eqs.(10) and (11) in eq.(7), one obtains

$$(V_1 - \lambda^2 p^2 T_1) = 0 \quad (12)$$

where

$$V_1 = \int_{\eta=0}^b \int_{\xi=0}^a (1 + \beta_1 (\xi/a))^3 (1 + \beta_2 (\eta/b)^2)^3 \left[W_{,\xi\xi}^2 - 4 \left(\frac{a}{b} \right) \sin \theta W_{,\xi\xi} W_{,\xi\eta} + 2 \left(\frac{a}{b} \right)^2 (\sin^2 \theta + \nu \cos^2 \theta) W_{,\xi\xi} W_{,\eta\eta} + 2 \left(\frac{a}{b} \right)^2 (1 + \sin^2 \theta - \nu \cos^2 \theta) W_{,\xi\eta}^2 - 4 \left(\frac{a}{b} \right)^3 \sin \theta W_{,\xi\eta} W_{,\eta\eta} + \left(\frac{a}{b} \right)^4 W_{,\eta\eta}^2 \right] d\xi d\eta \quad (13)$$

and

$$T_1 = \cos^4 \theta \int_{\eta=0}^b \int_{\xi=0}^a (1 + \beta_1 (\xi/a))(1 + \beta_2 (\eta/b)^2) W^2 d\xi d\eta \quad (14)$$

Here,

$$\lambda^2 = \frac{12a^4 \rho (1-\nu^2)}{Eh_0^2} \quad (15)$$

is a constant.

But equation (12) involves the unknown A_1 and A_2 arising due to the substitution of $W(\xi, \eta)$ from eq (9). These two constants are to be determined from eq (12), as follows:

$$\partial(V_1 - \lambda^2 p^2 T_1) / \partial A_n = 0, n = 1, 2 \quad (16)$$

Equation (16) simplifies to the form

$$b_{n1} A_1 + b_{n2} A_2 = 0, n = 1, 2 \quad (17)$$

where b_{n1} , b_{n2} ($n = 1, 2$) involve parametric constants and the frequency parameter.

For a non-trivial solution, the determinant of the coefficient of equation (17) must be zero. So one gets the frequency equation as

$$\begin{vmatrix} b_{11} & b_{12} \\ b_{21} & b_{22} \end{vmatrix} = 0 \quad (18)$$

where,

$$\begin{aligned} b_{11} &= (F_1 - \lambda^2 p^2 B_1), \\ b_{12} &= b_{21} = (F_2 - \lambda^2 p^2 B_2), \\ \text{and } b_{22} &= (F_3 - \lambda^2 p^2 B_3) \end{aligned}$$

Here $F_1, F_2, F_3, B_1, B_2, B_3$ involves parametric constants, skew angle and aspect ratio.

From eq. (18), one can obtain a quadratic equation in p^2 from which the two values of p^2 can be found. After determining A_1 & A_2 from eq.(17), one can obtain deflection function W . Choosing $A_1=1$, one obtains $A_2=(-b_{11}/b_{12})$ and then W comes out as

$$W = [XY(a/b)(1-X)(1-Y a/b)]^2 \times [1 + (-b_{11}/b_{12})XY(a/b)(1-X)(1-Y a/b)] \quad (19)$$

4. Differential Equation of Time Function and its Solution

Time functions of free vibrations of visco-elastic plates are defined by the general ordinary differential equation (3). Their form depends on the visco-elastic operator $\check{D}$.

For Kelvin's model, one has

$$\check{D} = \{1 + (\check{n}/G)(d/dt)\} \quad (20)$$

where, $\check{n}$ is visco-elastic constant and G is shear modulus.

The governing differential equation of time function of an parallelogram plate of variable thickness, by using eq.(20) in eq.(3), one obtains

$$T_{,tt} + p^2 (\check{n}/G) T_{,t} + p^2 T = 0 \quad (21)$$

Eq. (21) is a differential equation of order two for time function T .

Solution of eq.(21) comes out as

$$T(t) = e^{kt} [C_1 \cos(k_1 t) + C_2 \sin(k_1 t)] \quad (22)$$

where,

$$k = -p^2 \check{n}/2G \quad (23)$$

and

$$k_1 = p \left\{ 1 - (p \check{n}/2G)^2 \right\}^{1/2} \quad (24)$$

Let us take initial conditions as

$$T = 1 \quad \text{and} \quad dT/dt = 0 \quad \text{at} \quad t = 0 \quad (25)$$

Using initial conditions from eq.(25) in eq.(22), one obtains

$$T(t) = e^{kt} [\cos(k_1 t) + (-k/k_1) \sin(k_1 t)] \quad (26)$$

Thus, deflection w may be expressed, by using eq.(26) and (19) in eq.(4), to give

$$\begin{aligned} W &= [XY(a/b)(1-X)(1-Y a/b)]^2 \\ &\times [1 + (-b_{11}/b_{12})XY(a/b)(1-X)(1-Y a/b)] \\ &\times \left\{ e^{kt} [\cos(k_1 t) + (-k/k_1) \sin(k_1 t)] \right\} \end{aligned} \quad (27)$$

Time period of vibration of the plate is given by

$$K = 2\pi/p \quad (28)$$

where p is frequency given by eq.(18).

5. Results and Discussion

Time period and deflection are computed for visco-elastic parallelogram plate whose thickness varies linearly in one direction and parabolically in another direction for different value of skew angle(θ), taper constants (β_1 and β_2), and aspect ratio(a/b) at different points for first two mode of vibration. The material parameters have been taken as [17]: $E=7.08 \times 10^{10} \text{ N/M}^2$, $G=2.682 \times 10^{10} \text{ N/M}^2$, $\check{n}=1.4612 \times 10^6 \text{ N.S/M}^2$, $\rho=2.80 \times 10^3 \text{ Kg/M}^3$, $\nu=.345$ and $h_0=0.01$ meter.

All the results are presented in the tables 1-21. The value of time period (K) for $\beta_1=0.4$, $\beta_2=0.6$ and $\theta=45^\circ$ have been found to decrease 36.4152960% for first mode and 35.4467079% for second mode in comparison to rectangular plate at fixed aspect ratio ($a/b=1.5$).

The value of time period (K) for $\beta_1=0.4$, $\beta_2=0.6$ and $\theta=45^\circ$ have been found to decrease 30.2696427% for first mode and 29.7142383% for second mode in comparison to parallelogram plate of uniform thickness at fixed aspect ratio ($a/b=1.5$).

Table 1 show the results of time period(K) for different values of taper constant (β_1) and fixed aspect ratio ($a/b=1.5$) for different values of skew angle (θ) and taper constant (β_2) i.e. $\beta_2=0.0, \theta=0^\circ$; $\beta_2=0.0, \theta=45^\circ$; $\beta_2=0.6, \theta=0^\circ$; and $\beta_2=0.6, \theta=45^\circ$ for first two mode of vibration. It can be seen that the time period (K) decrease when taper constant (β_1) increase for the first two mode of vibration.

Table 2 show the results of time period(K) for different values of values of skew angle (θ) and fixed aspect ratio ($a/b=1.5$) for values taper constant (β_1 and β_2) i.e. $\beta_1=0.0, \beta_2=0.0$; $\beta_1=0.4, \beta_2=0.0$; $\beta_1=0.0, \beta_2=0.6$; and $\beta_1=0.4, \beta_2=0.6$ for first two mode of vibration. It can be seen that the time period (K) decrease when skew angle (θ) increase for two mode of vibration. Table 3 show the results of time period(K) for different values of taper constant (β_2) and fixed aspect ratio ($a/b=1.5$) for values of skew angle (θ) and taper constant (β_1) i.e. $\beta_1=0.0, \theta=0^\circ$; $\beta_1=0.0, \theta=45^\circ$; $\beta_1=0.6, \theta=0^\circ$; and $\beta_1=0.6, \theta=45^\circ$ for first two mode of vibration. It can be seen that the time period (K) decrease when taper constant (β_2) increase for two mode of vibration.

Table 4 and Table 5 show the results of time period(K) for different values of values of aspect ratio (a/b) and fixed angle ($\theta=0^\circ$ and 45°) for values taper constant (β_1 and β_2) i.e. $\beta_1=0.0, \beta_2=0.0$; $\beta_1=0.4, \beta_2=0.0$; $\beta_1=0.0, \beta_2=0.6$; and $\beta_1=0.4, \beta_2=0.6$ for first two mode of vibration. It can be seen that the time period (K) decrease when aspect ratio (a/b) increase for two mode of vibration.

Table 1. Time period K (in seconds) for different taper constant (β_1) and aspect ratio(a/b=1.5)

β_1	$\beta_2=0.0, \theta=0^\circ$		$\beta_2=0.6, \theta=0^\circ$		$\beta_2=0.0, \theta=45^\circ$		$\beta_2=0.6, \theta=45^\circ$	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.0	0.143648	0.037357	0.122005	0.032117	0.091306	0.024076	0.076959	0.020522
0.2	0.129973	0.033783	0.110399	0.028988	0.082649	0.021785	0.069919	0.018619
0.4	0.117855	0.030592	0.100131	0.026214	0.075015	0.019754	0.063668	0.016922
0.6	0.107322	0.027804	0.091211	0.023802	0.06839	0.017984	0.058209	0.015436
0.8	0.098231	0.025389	0.083514	0.021719	0.062672	0.016451	0.053470	0.014143

Table 2. Time period K (in seconds) for different angle (θ) and aspect ratio(a/b=1.5)

θ	$\beta_1=0.0, \beta_2=0.0$		$\beta_1=0.0, \beta_2=0.6$		$\beta_1=0.4, \beta_2=0.0$		$\beta_1=0.4, \beta_2=0.6$	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0°	0.143648	0.037357	0.122005	0.032117	0.117855	0.030592	0.100131	0.026214
15°	0.138373	0.036062	0.117457	0.030980	0.113546	0.029538	0.096603	0.025354
30°	0.121441	0.031827	0.102866	0.027271	0.099700	0.026087	0.084833	0.022397
45°	0.091306	0.024076	0.076959	0.020522	0.075015	0.019754	0.063668	0.016922
60°	0.051489	0.013611	0.042927	0.011481	0.042344	0.011183	0.035618	0.009500
75°	0.014511	0.003831	0.011784	0.003152	0.011958	0.003157	0.009808	0.002617

Table 3. Time period K (in seconds) for different taper constant (β_2) and aspect ratio(a/b=1.5)

β_2	$\beta_1=0.0, \theta=0^\circ$		$\beta_1=0.4, \theta=0^\circ$		$\beta_1=0.0, \theta=45^\circ$		$\beta_1=0.4, \theta=45^\circ$	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.0	0.143648	0.037357	0.117855	0.030592	0.091306	0.024076	0.075015	0.019754
0.2	0.135950	0.035437	0.111551	0.029015	0.086221	0.022785	0.071015	0.018747
0.4	0.128730	0.033681	0.105639	0.027551	0.081429	0.021594	0.067224	0.017798
0.6	0.122005	0.032117	0.100131	0.026214	0.076959	0.020522	0.063668	0.016922
0.8	0.115765	0.030742	0.095019	0.025003	0.072813	0.019572	0.060354	0.016121

Table 4. Time period K (in seconds) for different aspect ratio (a/b) and angle($\theta=0^\circ$)

a/b	$\beta_1=0.0, \beta_2=0.0$		$\beta_1=0.0, \beta_2=0.6$		$\beta_1=0.4, \beta_2=0.0$		$\beta_1=0.4, \beta_2=0.6$	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.5	0.172915	0.042600	0.147303	0.036965	0.141397	0.034747	0.120491	0.030027
1.0	0.160377	0.040555	0.136621	0.035085	0.131331	0.033132	0.111906	0.028556
1.5	0.143648	0.037357	0.122005	0.032117	0.117855	0.030592	0.100131	0.026214
2.0	0.125747	0.033392	0.106036	0.028431	0.103384	0.027418	0.087225	0.023275
2.5	0.108496	0.029151	0.090545	0.024526	0.089389	0.023998	0.074656	0.020130

Table 5. Time period K (in seconds) for different aspect ratio (a/b) and angle($\theta=45^\circ$)

a/b	$\beta_1=0.0, \beta_2=0.0$		$\beta_1=0.0, \beta_2=0.6$		$\beta_1=0.4, \beta_2=0.0$		$\beta_1=0.4, \beta_2=0.6$	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.5	0.119167	0.029503	0.101497	0.025586	0.097468	0.024071	0.083302	0.020867
1.0	0.106813	0.027355	0.090789	0.023591	0.087533	0.022371	0.074825	0.019351
1.5	0.091306	0.024076	0.076959	0.020522	0.075015	0.019754	0.063668	0.016922
2.0	0.075799	0.020293	0.063004	0.017022	0.062443	0.016705	0.052277	0.014083
2.5	0.062002	0.016658	0.050771	0.013742	0.051201	0.013750	0.042206	0.011387

The value of deflection (w) for $\beta_1=0.4, \beta_2=0.6$ and $\theta=45^\circ$ have been found to increase 5.2670080% for first mode and no effect for second mode in comparison to parallelogram plate of uniform thickness for initial time 0.K at X=0.2, Y=0.4 and aspect ratio(a/b=1.5).

The value of deflection (w) for $\beta_1=0.4, \beta_2=0.6$ and $\theta=45^\circ$ have been found to decrease 2.7045301% for first mode and 0.2659574% for second mode in comparison to rectangular plate for initial time 0.K at X=0.2, Y=0.4 and a/b=1.5.

The value of deflection (w) for $\beta_1=0.4, \beta_2=0.6$ and $\theta=45^\circ$ have been found to increase 1.8912530% for first mode and decrease 10.9929078% for second mode in comparison to parallelogram plate of uniform thickness for time 5.K at X=0.2, Y=0.4 and a/b=1.5.

The value of deflection (w) for $\beta_1=0.4, \beta_2=0.6$ and $\theta=45^\circ$ have been found to decrease 6.3721941% for first

mode and 13.4482759% for second mode in comparison to rectangular plate for time 5.K at X=0.2, Y=0.4 and a/b=1.5.

Tables 6 - 21 shows the results of deflection (w) for different values of X, Y and fixed taper constant ($\beta_1=0.0, 0.4$ and $\beta_2=0.0, 0.6$), and aspect ratio (a/b=1.5) for two values of angle (θ) i.e. $\theta=0^\circ$ and $\theta=45^\circ$ for first two mode of vibration with time 0.K and 5.K. It can be seen that deflection (w) start from zero to increase then decrease to zero for first two mode of vibration (except second mode at Y= 0.2 and 0.4) and second mode of vibration deflection (w) at Y=0.2 start zero to increase then decrease then increase then decrease and finally become to zero and second mode of vibration deflection (w) at Y=0.4 start zero to decrease then increase and finally become to zero for different value of X.

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Table 19. Deflection w for different X, Y and $\beta_1=0.0$, $\beta_2=0.6, \theta=45^\circ$ and $a/b=1.5$ at time 5.K

X	Y=0.2		Y=0.4		Y=0.6		Y=0.8	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000
0.2	0.000995	0.000281	0.001293	0.000269	0.000187	0.000107	0.001406	0.001847
0.4	0.002197	0.000038	0.002846	-0.000284	0.000417	0.000194	0.003227	0.005045
0.6	0.002197	0.000038	0.002846	-0.000284	0.000417	0.000194	0.003227	0.005045
0.8	0.000995	0.000281	0.001293	0.000269	0.000187	0.000107	0.001406	0.001847
1.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000

Table 20. Deflection w for different X, Y and $\beta_1=0.4$, $\beta_2=0.0, \theta=45^\circ$ and $a/b=1.5$ at time 5.K

X	Y=0.2		Y=0.4		Y=0.6		Y=0.8	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000
0.2	0.000981	0.000278	0.001272	0.000266	0.000185	0.000106	0.001421	0.001823
0.4	0.002151	0.000039	0.002778	-0.000277	0.000413	0.000192	0.003281	0.004977
0.6	0.002151	0.000039	0.002778	-0.000277	0.000413	0.000192	0.003281	0.004977
0.8	0.000981	0.000278	0.001272	0.000266	0.000185	0.000106	0.001421	0.001823
1.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000

Table 21. Deflection w for different X, Y and $\beta_1=0.4$, $\beta_2=0.6, \theta=45^\circ$ and $a/b=1.5$ at time 5.K

X	Y=0.2		Y=0.4		Y=0.6		Y=0.8	
	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode	First Mode	Second Mode
0.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000
0.2	0.000993	0.000262	0.001293	0.000251	0.000185	0.000100	0.001357	0.001721
0.4	0.002209	0.000036	0.002872	-0.000262	0.000413	0.000181	0.003090	0.004698
0.6	0.002209	0.000036	0.002872	-0.000262	0.000413	0.000181	0.003090	0.004698
0.8	0.000993	0.000262	0.001293	0.000251	0.000185	0.000100	0.001357	0.001721
1.0	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000	0.000000

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